# Numerical study of flushing half-cone formation due to pressurized sediment flushing

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ABSTRACT: Pressure flushing cleans up the hydropower plant entrance from the sediment deposition. This technique results in localized scouring, that is called flushing cone. In this study, the flushing cone formation is simulated by implementing a RANS based 3D numerical model that consists of two successive stages: hydraulic flow and sediment erosion simulation. The velocity distribution profiles and the final geometry of the flushing cone were used to verify the numerical model outcomes. The results obtained by the hydraulic flow simulation revealed that the calculated velocity distribution profiles were comparable with the experimental data and the universal wall equations with roughness,  $k_s = 10$  mm as a sediment erosion process using the moving boundary method the model should be improved for better accuracy in predicting flushing cone geometry.

# 1 INTRODUCTION

One of most pressing problems of river and dam management in current times is the excess deposition of sediment. Severe sediment deposition which occurs in dam reservoirs decreases the water storage capacity and thus reduces the efficiency of the dam operation related to flood control and flow regulation (Graf, 1984). On top of the issues directly related to sedimentation in dam reservoirs, including intake clogging and machinery abrasion, a number of environmental problems also arise downstream of the dam. These include riverbed and bank erosion, armouring of the river bed, and a reduction in the quality of habitat for aquatic species, thereby threatening reductions in river system biodiversity (Kondolof, 1997).

To negate the issues related to sedimentation, a number of techniques have been proposed such as density current venting, flushing, by passing, dredging and sluicing (Brandt, 2000). However, flushing has proven to be one of the most effective and efficient in removing reservoir sediment through the bottom outlet (Shen and Lai, 1996). This technique can be categorized into two types; free flushing and pressure flushing. In free flushing the water level is drawn down to a relatively low level which causes large scale scouring in the form of a channel along the reservoir. Whereas, in pressure flushing the water level is maintained at a near constant level high above the bottom outlet such that the loss of water, an important dam resource, is conserved. Here, the scour formation shape is similar to that of a half-cone, often simply referred to as a flushing cone. This paper is concerned with pressure flushing and the formation of the cone.

Previously a few studies have experimentally investigated the flushing cone formation and proposed empirical equations to estimate the final volume and length of flushing cone (Scheuerlein et al., 2004, Emangholizadeh, 2005, Meshkati et al., 2010, and Fathi-Moghadam et al., 2010). While empirical regression-based relations can allow for a quick and simple estimate of final cone geometry, these equations are often flawed because of the high nonlinearity between the cone geometry and the significant parameters.

Instead of using experimental and analytical methods, a less costly and potentially more accurate alternative is to use a numerical model (Olsen, 1999). A number of numerical studies have been conducted to simulate the flushing operation in dam reservoirs. Most of these studies are devoted to either 1D or 2D numerical models. Holly and

Rahuel (1990); Lai and Shen (1996) and Basson and Rooseboom (1997), individually developed a 1-D model to simulate the bed topography change during the drawdown phase in free flushing. These models may only be used to study flushing processes in long and narrow reservoirs. Ruland and Rouve (1992) utilized a 2D finite element model to simulate the erosion process in the reservoir. Olsen (1999) solved the depth-averaged Navier-Stokes equation on a 2D mesh and extrapolated the corresponding flow field onto a 3D mesh to solve sediment concentration in flushing processes. Although, in addition to its difficult analytical characterization, the formation of the flushing cone is a complex three-dimensional phenomenon (Scheuerlein et al., 2004), where turbulent eddies, secondary currents and vertical velocities are significant, particularly in the vicinity of the bottom outlet. However, as can be found in the literature only a few studies have considered a 3D numerical model and the majority of these are devoted to only free flushing phenomena, e.g.: Haun and Olsen (2012a, b) where a RANS model that solves the transient-convection equation for suspended sediment transport and Van Rijn's (1984) empirical formula for bed load transport was utilized.

In this study, the formation of the flushing cone is simulated utilizing a RANS based 3D numerical model verified by Yoneyama (2010) and Yoneyama et al. (2012a, b) and adapted for this analysis. The simulation process consists of two stages. In the first stage the velocity distribution and equilibrium bed topography (final shape) of the flushing half-cone, from the experimental investigation conducted by Emamgholizadeh (2005), were used for verification and calibration of the hydraulic model. In the second stage the numerical model commences from the initial sediment condition incorporating the moving boundary method to simulate the sediment erosion process. The moving boundary method, while primitive, was chosen because of its simplicity and since only the final bed shape is of interest in this analysis. The results of the simulation in the second phase are compared to the geometry of the flushing cone as found experimentally.

# 2 EXPERIMENTATION

#### 2.1 Experimental setup

A hexahedral shallow basin of size 3.0 m length, 1.5 m width and 1.5 m height was used as the reservoir model. A schematic side and plan view of the model as well as the scoured flushing cone in the vicinity of bottom outlet are shown in Figure 1. A gate valve was installed in the center



Figure 1. Schematic side and plan view of the reservoir model, the flushing cone and its relevant hydraulic parameters.

of the front wall of the model (dam wall) as a representation of the bottom outlet. A 50.8 mm diameter gate valve was employed. Inside the reservoir, sediment was placed and paved in the reach covering the entire width and 2.0 m of the length from the dam wall. The sediment thickness was 0.32 m, measured from the lower invert of the outlet to the sediment surface. Sand particles with a uniform size distribution were used as sediment. The particles had a median diameter, d<sub>50</sub> of 1.2 mm and specific gravity, G<sub>s</sub> of 2.65. Inflow discharges of 1.0, 3.0, 4.5, 6.0 and 8.0 L/s were used at the upstream end. The discharge at the outlet was kept the same as that of the inflow so that the reservoir water level was maintained constant at the required elevation. Three constant water levels of 0.425, 0.825 and 1.125 m were examined.

A digital point gauge device was utilized to measure the scour cone configuration after each experiment. The vertical velocity profile was measured along six different axis locations by Laser-Doppler (LDV). The axis locations were selected along the x-axis (centerline of the bottom outlet), separated by equal 0.1 m intervals and numbered I–VI in Figure 1.

## 2.2 Experimental procedure and design

The following general procedure was enacted for all tests conducted in the experimental study. Before each test the sediment was wetted, flattened and leveled to a specific level above the lower invert of the bottom outlet,  $H_s$ . To begin the test run, the model was slowly filled with water by adjusting the incoming flow rate into the reservoir. When the water surface elevation in the reservoir above the lower invert of the outlet,  $H_w$  reached the desired level, the bottom outlet was carefully opened until the outflow discharge became equal to that of the inflow discharge. Subsequently, the sediment flushing began, flowing out from the reservoir into the settling basin. Experiments were continued until equilibrium conditions prevailed in which the sediment concentration released from the reservoir was negligible, signaling the end of the experiment. The development of the flushing cone was fairly quick with the bulk of the formation finishing within a few minutes or less.

At this stage, the vertical velocity profiles at the six axes were measured using LDV. Afterwards, at the end of each test case as soon as the pump was turned off and the incoming discharge was set to zero, the outlet was closed immediately. Thereafter, water was carefully and slowly drained from the reservoir via a pump. The final scour contour was measured using digital point gauges. The volume of the flushing cone was calculated by Surfer (8.0) Software.

#### **3 NUMERICAL METHOD**

#### 3.1 Basic equations

The numerical method consists of the RANS equations combined with the FAVOR method (Hirt & Sicilian, 1985) for modelling the interface between fluid and obstacles, and the VOF model (Hirt & Nichols, 1985) for modelling the free surface. Then, the continuity, momentum and advection of fluid equations 1–3 in this form are written as;

Continuity:

$$\frac{\partial \gamma^{\nu}}{\partial t} + \frac{\partial \gamma^{\mu}_{j} \overline{u}_{j}}{\partial x_{j}} = 0$$
(1)

Momentum (i = 1, 2, 3):

$$\frac{\partial \overline{u}_i}{\partial t} + \frac{\gamma_j^{\alpha} \overline{u}_j}{\gamma^{\nu}} \frac{\partial \overline{u}_i}{\partial x_j} = g_i - \frac{1}{\rho} \frac{\partial \overline{p}}{\partial x_i} + \frac{\partial}{\partial x_j} \left( \nu \frac{\partial \overline{u}_i}{\partial x_j} - \overline{u_i u_j} \right)$$
(2)

Advection of fluid:

$$\frac{\partial \gamma^{\nu} F}{\partial t} + \frac{\partial \gamma_{j}^{a} F \overline{u}_{j}}{\partial x_{j}} = 0$$
(3)

where  $\gamma^{v}$  is the void ratio of a cell,  $\gamma_{i}^{a}$  is the aperture ratio of a cell boundary,  $u_{i}$  is the flow velocity component,  $g_{i}$  is the external force per unit volume, p is the pressure,  $\rho$  is the density of the fluid, v is the dynamic viscosity, F is the fluid volume fraction of a cell, - is the Reynolds averaging quantity and ' is the fluctuation in the Reynolds average. The k- $\varepsilon$  turbulence closure model is utilized to form equations 4–6 to model the Reynolds stress term,  $-u_{i}u_{i}$ :

$$-\overline{u_i \, u_j} = \frac{C_{\mu} k^2}{\varepsilon} \left( \frac{\partial \overline{u_i}}{\partial x_j} + \frac{\partial \overline{u_j}}{\partial x_i} \right) - \frac{2}{3} k \delta_{i,j} \tag{4}$$

$$\frac{\partial k}{\partial t} + \frac{1}{\gamma^{\nu}} \frac{\partial \gamma_{j}^{\mu} k \overline{u}_{j}}{\partial x_{j}} = \frac{\partial}{\partial x_{j}} \left[ \left( \nu + \frac{C_{\mu} k^{2}}{\sigma_{k}^{\varepsilon}} \right) \frac{\partial k}{\partial x_{j}} \right] - \overline{u_{i}} \overline{u_{j}} \frac{\partial \overline{u_{i}}}{\partial x_{j}} - \varepsilon$$
(5)

$$\frac{\partial \varepsilon}{\partial t} + \frac{1}{\gamma^{\nu}} \frac{\partial \gamma_{j}^{a} \varepsilon \overline{u}_{j}}{\partial x_{j}} = \frac{\partial}{\partial x_{j}} \left[ \left( \nu + \frac{C_{\mu} k^{2}}{\sigma_{\varepsilon}^{\varepsilon}} \right) \frac{\partial \varepsilon}{\partial x_{j}} \right] \\ - C_{\varepsilon^{1}} \frac{\varepsilon}{k} \frac{\overline{u}_{i} u_{j}}{u_{j}} \frac{\partial \overline{u}_{i}}{\partial x_{j}} - C_{\varepsilon^{2}} \frac{\varepsilon^{2}}{k}$$
(6)

where  $\underline{k} \equiv -\overline{\dot{u}_i \dot{u}_j}/2$  is the turbulent kinetic energy,  $\varepsilon \equiv v \dot{u}_{i,j} \dot{u}_{i,j}$  is the turbulent energy dissipation, and  $\delta_{i,j}$  is the Kronecker delta. The constants in equations 3–5 are  $\sigma_k = 1.0$ ,  $\sigma_{\varepsilon} = 1.3$ ,  $C_{\mu} = 0.09$ ,  $C_{\varepsilon l} = 1.45$ , and  $C_{\varepsilon 2} = 1.92$ . The discretized form of equations 1 and 2 are solved using the SIMPLE method (Patankar et al., 1972) on a staggered grid system. Finally, while in this study the free surface is not such a significant parameter, the VOF method is used for its calculation.

#### 3.2 Moving boundary method

As shown in equations 1 and 2, the FAVOR method is used to model the interface between the fluid and obstacles; in this case sediment. Here the void ratio,  $\gamma^{v}$  and aperture ratios,  $\gamma^{a_{i}}$  are calculated to indicate the proportion and orientation of fluid in a cell. Thus, we can view the interface between the sediment and fluid (in one plane) as consisting of two nodes and a straight boundary line as illustrated in Figure 2, with the corresponding void and aperture ratios indicated.

The magnitude of the tangential velocity just above the boundary node,  $U_{tang}$  is compared with the critical velocity for sediment transport,  $U_{cr}$ . When  $U_{tang}$  is larger than  $U_{cr}$ , the boundary node is moved down at a specified velocity,  $V_d$ , as illustrated



Figure 2. Calculation cell containing both fluid and sediment, with its corresponding void ratio,  $\gamma^{\nu}$  and aperture ratios,  $\gamma^{a}_{\nu}$ .



Figure 3. Movement of the boundary node when the critical velocity for sediment transport is exceeded near the node.



Figure 4. Movement of the boundary node towards the angle of repose in submerged conditions,  $\alpha_s$ .

in Figure 3. Furthermore, the angle of the boundary line cannot exceed the angle of repose in submerged conditions,  $\alpha_s$ . In this case the higher of the two boundary nodes are lowered at the velocity,  $V_d$ , until the angle of repose is reached (Fig. 4).

# 4 NUMERICAL CONDITIONS

# 4.1 Domain

The domain size and conditions are the same as the experiments conducted by Emangholizadeh (2005).

The length is 3.0 m, width 1.5 m and maximum domain height of 1.0 m. The cell size in the x-direction ranges from 0.05 m at the inlet end, to 0.02 m in the area close to the outlet. The size of the cell in the y-direction is 0.025 m at the sides and as small as 0.01 m at the outlet. Similarly the cell size in the vertical direction is 0.01 m near the outlet and up to 0.02 m elsewhere. A total of around 390,000 cells are used to make up the domain but approximately 220,000 cells are used in calculation. Figure 5 provides a snapshot of the calculation domain with an illustration of the parameters Q,  $H_w$  and  $H_s$ .

#### 4.2 Bottom outlet representation

The bottom outlet was represented as a polygon made up of the rectangular cells of the numerical mesh as illustrated in Figure 6. The maximum cross-sectional width of the numerical outlet was 50 mm (compared with a diameter of 50.8 mm for the experiments) and had a total cross-sectional area of  $2.0 \times 10^{-3}$  m<sup>2</sup> (compared with  $2.03 \times 10^{-3}$  m<sup>2</sup> for the experiments). The bottom outlet was modeled by manipulating the streamwise aperture ratio,  $\gamma^{a}_{x}$  into the last cell in the calculation domain as indicated in Figure 6.



Figure 5. Side-view snapshot of the velocity profile in experiment Case D (1st stage) with parameters Q,  $H_w$  and  $H_s$  shown.



Figure 6. Diagram of the bottom outlet representation in the YZ plane, and its aperture ratios,  $\gamma^a_{x}$ .

# 4.3 Boundary conditions

Both the upstream inlet boundary and the downstream bottom outlet boundary were subjected to Dirichlet boundary conditions.

At the inlet, both the constant water elevation,  $H_w$ , and the resulting  $u_x$  velocity component calculated from the input flow-rate, Q, over the crosssectional area were specified. Since a single cell contains two cell boundaries in the x-direction, the velocity,  $u_x$  was specified for both cell boundaries. The velocity distribution was uniform.

The  $u_x$  velocity component at the bottom outlet was specified by the setting the outflow equal to the inlet flow rate, Q. This flow rate was divided by the cross-sectional area of the bottom outlet, found in section 4.2, to calculate and set the same  $u_x$  velocity component at each cell in the bottom outlet. In addition, since the inflow takes some time to reach and affect the area downstream, a "warming-up period" of 2.5s was specified to avoid imparting an initial rapid outflow velocity with no inflow into the control volume. Within this period, the outflow velocity was linearly increased from zero at 0s, to the calculated velocity based on the inflow, Q at 2.5s.

#### 4.4 Numerical cases

Table 1 shows the experimental conditions that were used for simulation.

Each case was made up of two stages. The first stage consisted of solely a hydraulic study with the sediment, represented as a rigid bed, at the final stage as measured from the experiments. The resulting velocity profile was compared with that of the experiments. The simulation was calibrated by adjusting the boundary condition at the bed. The second stage took the calibrated hydraulic model and incorporated the moving boundary method. The initial condition was that of the sediment placed flat at a uniform depth throughout the entire domain. The simulation was run until the model predicted no further change in scour shape, and compared with the measured experimental scour shape (the flushing cone).

 Table 1. Experimental conditions applied to numerical study.

Case	Q (L/s)	$H_w$ (m)	$H_s$ (m)	D (cm)	<i>d</i> <sub>50</sub> (mm)
A	6.0	0.425	0.32	5.08	1.2
В	8.0	0.425	0.32	5.08	1.2
С	6.0	0.825	0.32	5.08	1.2
D	8.0	0.825	0.32	5.08	1.2
Е	6.0	1.125	0.32	5.08	1.2
F	8.0	1.125	0.32	5.08	1.2

#### 5 NUMERICAL RESULTS

#### 5.1 *Rigid bed analysis*

In this stage of the study various sediment boundary conditions were tried to identify the best representation of the experimental work. The simplest boundary conditions to apply are the no-slip, halfslip or free-slip conditions. The other sediment boundary conditions applied were the universal wall equations. These were applied in empirical form and as suggested by Le Roux (2004) which gives equations for all three hydrodynamic stages (rough, smooth and transitional). These equations were adjusted by changing the roughness of the rigid bed,  $k_s$ . The sediment boundary conditions are shown in Table 2.

The selection of  $k_s = 1.0$  mm is because it is very close to the mean sediment diameter while  $k_s = 10$  mm represents a much rougher condition as a comparison. Figures 7–10 show velocity profiles at axis 1 and axis 2 for Cases B and D.

Typically, the velocities at low heights above the sediment surface were either; overestimated near the bottom outlet (axis 1) except for the free-slip case or underestimated further from the bottom outlet (axis 2) in the calculation. For larger heights above the sediment surface better agreement was

Table 2. Sediment boundary conditions applied.

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Condition	Туре	$k_s$ (mm)			
Ι	Free-slip	_			
II	No-slip	_			
III	Half-slip	_			
IV	Wall equation	1.0			
V	Wall equation	10.0			



Figure 7. Velocity profile comparison at axis 1 for Case B.



Figure 8. Velocity profile comparison at axis 1 for Case D.



Figure 9. Velocity profile comparison at axis 2 for Case B.



Figure 10. Velocity profile comparison at axis 2 for Case D.

found between the experimental and calculation data, however slightly underestimated in general. To obtain better results the boundary condition at the bottom outlet may need to be modelled differently. For example, by using a pressure boundary condition that sets the pressure of the outflow equal to atmospheric the outflow will depend on the hydrostatic pressure in the reservoir rather than the numerically stipulated outflow velocity calculated using Q. Here, as the outflow increases so that it exceeds Q, the opening of the bottom outlet, as defined by the aperture ratios, can be decreased so that the outflow is equivalent to Q.

It was found that by comparing RMSE and R<sup>2</sup> curve fit values between the experimental data and the numerical data for all axes, considering only boundary conditions I-III, the no slip sediment boundary condition (II) produced more favourable results overall. When comparing with the wall equations (IV and V), only very tiny differences were evident between each other and the no-slip condition. This is because the flow is generally hydrodynamically smooth everywhere. Here, the roughness,  $k_s$ does not influence on the calculation of the velocity at the boundary. In fact only at the peak velocities of flow near the bottom outlet where the flow could be in a hydrodynamically transitional or rough stage are any notable differences detected between the wall equations and the no slip condition. But as noted above, the peak velocities at axis 1 are overestimated and can be reduced slightly by imposing the wall equations and a higher value of  $k_s$ . More investigation is required to determine the optimum value of  $k_s$  for this study but  $k_s = 10 \text{ mm produced}$ slightly improved results from  $k_s = 1$  mm.

#### 5.2 Movable bed analysis

In this stage the calculation is conducted with an initially uniform sediment bed level (0.32 m for all cases). The wall equations are applied as the sediment boundary condition determined from stage 1. The calculation is run until no notable change was evident between a sufficient time gap. Similar to the experiment this generally occurs after about 60-120s depending on the calculation case.

Figures 11 and 12 show the final bed topography for Cases B and D respectively. As can be seen a cone



Figure 11. Final flushing cone geometry for Case B.



Figure 12. Final flushing cone geometry for Case D.

shape is predicted however the simulation could not yet lead to robust results in terms of flushing volume and length. It is possible that an analysis considering bed load transport would allow for better results; however it is assumed that in pressure flushing most of the sediment is quickly removed through the bottom outlet without significant redeposition. The simulations are ongoing and by improving the bottom outlet and the sediment boundary conditions, better results using the moving boundary method are foreseeable in the near future.

#### 6 CONCLUSION

A numerical study to simulate the pressure flushing in dam reservoirs was conducted. An experimental study by Emamgholizadeh (2005) was used to verify the numerical analysis. Velocity profiles were shown to have moderate agreement with the experiments. By defining the bottom outlet velocity as a function of hydrostatic pressure, the simulation results may improve. Furthermore, it was found that the wall equations where  $k_s = 10$  mm as the sediment boundary condition lead to better agreement in general. The simulation by the moving boundary method to calculate the sediment removal is still undergoing and the calculation of its geometrical properties could not yet be obtained accurately. The authors are optimistic that work in the near future will attain amenable results by making improvements in the bottom outlet boundary condition, cell sizes and sediment boundary condition.

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